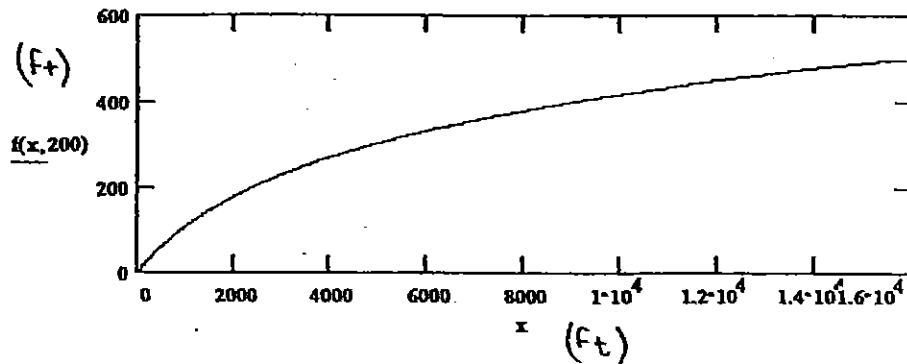


PROBLEM 2.1

CHAPTER 2

(a) $f(x) = 200 \ln [7.06 \times 10^{-4} (x + 1416)]$ ft.
Road Profile



(b) The position vector is:

$$\vec{r}(x) = x \vec{i} + 200 \ln [7.06 \times 10^{-4} (x + 1416)] \vec{j}$$

The roadway tangent vector is:

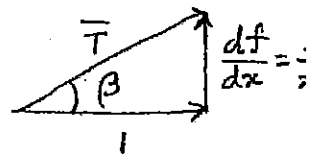
$$\vec{T}(x) = \frac{d\vec{r}(x)}{dx} = \vec{i} + \frac{200}{x + 1416} \vec{j}$$

$$\therefore \tan \beta = \frac{200}{x + 1416}$$

$$\Rightarrow \boxed{\beta(x) = \tan^{-1} \left(\frac{200}{x + 1416} \right)}$$

$$\beta(500 \text{ ft}) = \tan^{-1} \frac{200}{1916} = 5.9^\circ$$

$$\Rightarrow \boxed{\beta(500 \text{ ft}) = 5.9^\circ}$$



(c) % grade = $100 \tan \beta = 100 \times \frac{200}{x + 1416}$

$$\therefore \boxed{\% \text{ grade} = \frac{20000}{x + 1416}}$$

$$\boxed{\% \text{ grade}(500 \text{ ft}) = 10.4 \%}$$

$$(d) \quad s(x) = \int_{\tau=0}^x \|\bar{T}(\tau)\| d\tau$$

$$\text{where, } \|\bar{T}(\tau)\| = \sqrt{1 + \left(\frac{200}{\tau + 1416}\right)^2}$$

$$\text{Let } u = \tau + 1416$$

$$\Rightarrow du = d\tau$$

$$\text{Then, } s(x) = \int_{u=1416}^{u=x+1416} \sqrt{1 + \frac{200^2}{u^2}} du$$

$$= \int_{1416}^{x+1416} \frac{\sqrt{u^2 + 200^2}}{u} du$$

$$\Rightarrow s(x) = \left\{ \sqrt{u^2 + 200^2} - 200 \ln \left[\frac{200 + \sqrt{u^2 + 200^2}}{u} \right] \right\} \Bigg|_{1416}^{x+1416}$$

[FROM TABLE OF INTEGRALS]

$$\Rightarrow s(x) = \left[\sqrt{(x+1416)^2 + 200^2} - 200 \ln \left[\frac{200 + \sqrt{(x+1416)^2 + 200^2}}{x+1416} \right] \right] \\ - \left[\sqrt{1416^2 + 200^2} + 200 \ln \left[\frac{200 + \sqrt{1416^2 + 200^2}}{1416} \right] \right]$$

$$\Rightarrow \boxed{s(x) = \sqrt{(x+1416)^2 + 200^2} - 200 \ln \left[\frac{200 + \sqrt{(x+1416)^2 + 200^2}}{x+1416} \right]} \\ \boxed{-1402}$$

$$\boxed{s(500 \text{ ft.}) = 507 \text{ ft.}}$$

PROBLEM 2.2

$$\vec{r}(x) = x_f \vec{i}_f + 4.1\sqrt{x_f} \vec{j}_f$$

$$\vec{T}(x) = \frac{d\vec{r}}{dx} = \vec{i}_f + \frac{df}{dx} \vec{j}_f$$

$$(a) \quad \beta(x_f) = \tan^{-1} \frac{df}{dx_f} = \tan^{-1} \frac{2.05}{\sqrt{x_f}}$$

$$\beta(x_f) \Big|_{x_f=5280} = \tan^{-1} \frac{2.05}{\sqrt{5280}} = 1.62^\circ$$

$$(b) \quad \|\vec{T}\| = \sqrt{1 + \left(\frac{df}{dx_f}\right)^2}$$

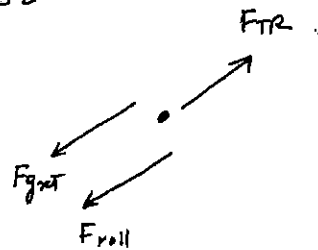
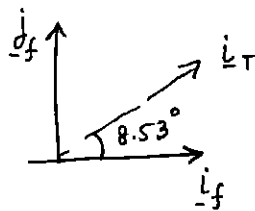
$$S = \int_0^{2\text{mi}} \|\vec{T}\| dx_f = \int_0^{5280} \sqrt{1 + \left(\frac{2.05}{\sqrt{x_f}}\right)^2} dx_f$$

THE INTEGRAL CAN BE SOLVED NUMERICALLY.

$$\Rightarrow S = 5300 \text{ ft.}$$

PROBLEM 2.3

$$(a) \quad \beta = \tan^{-1} \left(\frac{15}{100} \right) = 8.53^\circ$$



EV at rest

$$\Rightarrow \sum F = 0$$

$$\Rightarrow F_{TR} - F_{gxt} - F_{roll} = 0$$

$$(i) \text{ if } F_{roll} = 0 \Rightarrow F_{TR} = F_{gxt}$$

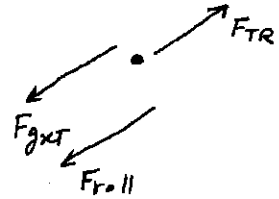
$$= mg \sin \beta = 692 \times 9.81 \times \sin 8.53$$

$$= 1007 \text{ N}$$

$$\Rightarrow F_{TR} = 1007 \text{ N}$$

(ii)

since the vehicle is at standstill,



$$F_{roll} = (F_{TR} - F_{gxt}) \quad \text{if } v_{xt} = 0 \quad \text{and} \quad |F_{TR} - F_{gxt}| \leq C_0 n$$

F_{roll} opposes the backward motion of the vehicle. Therefore, the maximum rolling resistance in this case is,

$$|F_{roll}| = C_0 mg$$

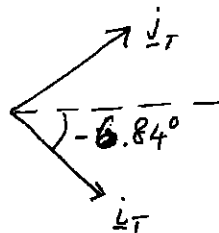
$$\text{and } F_{roll} = -C_0 mg$$

$$\text{Therefore, } F_{TR}|_{\min} - F_{gxt} = -C_0 mg$$

$$\Rightarrow F_{TR}|_{\min} = F_{gxt} - C_0 mg = 1007 - (0.009)(6789) = 946 \text{ N}$$

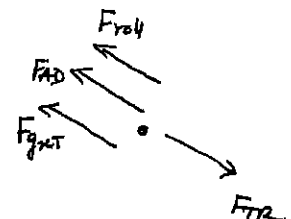
$F_{TR}|_{\min}$ is the minimum tractive force required to prevent the vehicle from rolling backwards.

$$(b) \quad \beta = \tan^{-1}\left(\frac{-12}{100}\right) = -6.84^\circ$$



$$F_{TR} = F_{gxt} + F_{AD} + F_{roll}$$

$$F_{AD} = \frac{P}{2} C_D A F v^2 = 0.232 v^2$$



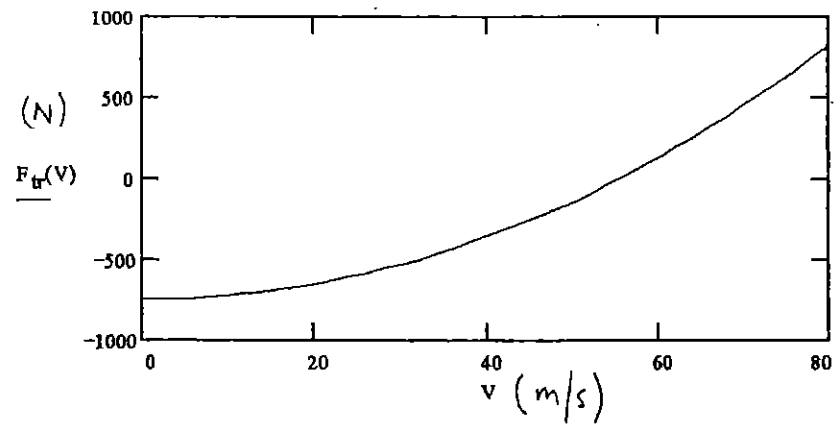
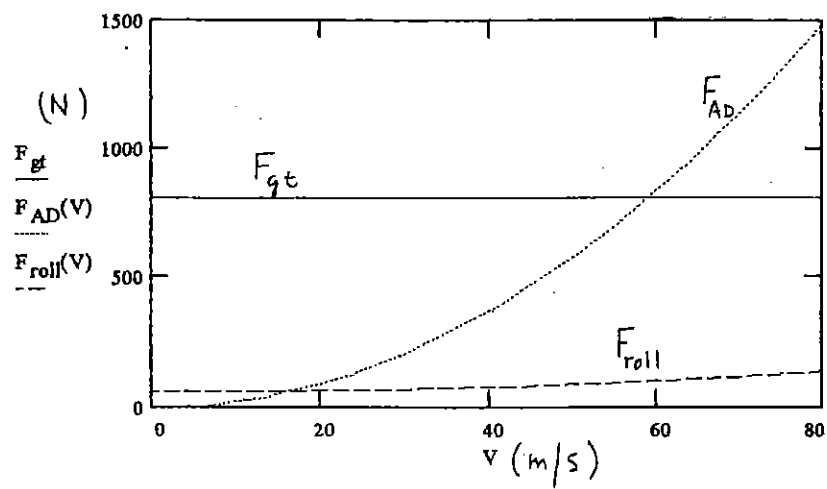
$$\begin{aligned} F_{gxt} &= mg \sin(\beta) \\ &= 692 \times 9.81 \sin(-6.84) \\ &= -809 \text{ N} \end{aligned}$$

$$\begin{aligned} F_{roll} &= mg (C_0 + C_1 v^2) \\ &= 6789 (0.009 + 1.75 \times 10^{-6} v^2) \\ &= 61.1 + 0.012 v^2 \end{aligned}$$

$$\Rightarrow F_{TR} = 0.232v^2 + 61.1 + .012v^2 - 809$$

$$\Rightarrow \boxed{F_{TR} = 0.244v^2 - 748} \text{ N}$$

PLOTS



PROBLEM 2.4

For $t > 0$

$$\Sigma F = m \frac{dv}{dt}$$

$$\Rightarrow F_{TR} - \operatorname{sgn}[v] \left[\frac{\rho}{2} C_D A_F \right] v^2 - \operatorname{sgn}(v) mg (C_0 + C_1 v^2) - mg \sin \beta = m \frac{dv}{dt}$$

$$\Rightarrow \frac{dv}{dt} = \left[\frac{F_{TR}}{m} - g (\sin \beta + C_0) \right] - \left[\frac{\rho}{2} C_D A_F + g C_1 \right] v^2$$

$[\operatorname{sgn}[v] = 1 \text{ since } v(t) > 0]$

Let $K_7 = \frac{F_{TR}}{m} - g (\sin \beta + C_0)$

$$K_8 = \frac{\rho}{2m} C_D A_F + g C_1$$

Note, $K_8 > 0$. Also $K_7 > 0$ since it is required that $\frac{dv(0)}{dt} > 0$ in order to satisfy $v(t) > 0$ for $t > 0$

And $\frac{dv(0)}{dt} > 0 \Rightarrow F_{g_{XT}} + F_{TR} > C_0 mg$

$$\frac{dv}{dt} = K_7 - K_8 v^2$$

$$\Rightarrow \frac{dv}{\frac{K_7}{K_8} - v^2} = K_8 dt \Rightarrow \int_{v(0)=0}^{v(t)} \frac{dv}{\left(\sqrt{\frac{K_7}{K_8}}\right)^2 - v^2} = K_8 \int_0^t dt$$

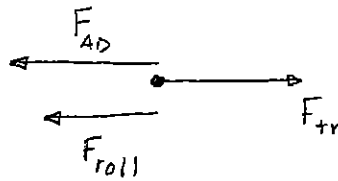
$$\Rightarrow v(t) \Rightarrow \frac{1}{\sqrt{\frac{K_7}{K_8}}} \tanh^{-1} \left(\sqrt{\frac{K_7}{K_8}} v \right) = K_8 t$$

$$\Rightarrow v(t) = V_T \tanh \left(\sqrt{K_7 K_8} t \right)$$

where $V_T = \sqrt{\frac{K_7}{K_8}}$

PROBLEM 2.5

(a)



$$F_{tr} - F_{AD} - F_{roll} = m \frac{dv}{dt}$$

$$F_{tr} = m \frac{dv}{dt} + F_{AD} + F_{roll}$$

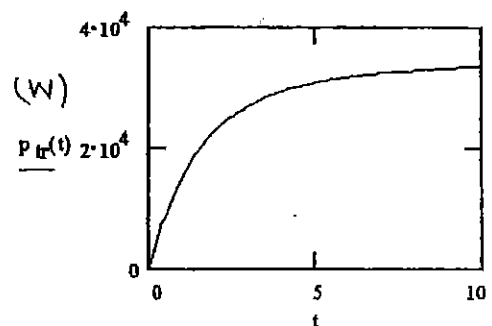
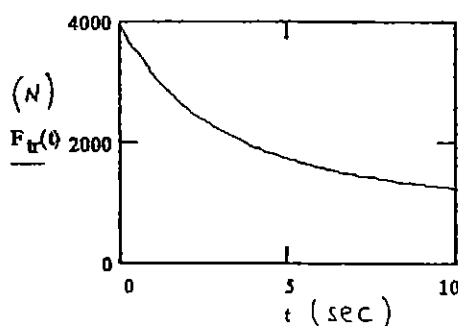
$$F_{tr}(t) = \frac{692(20)(0.282)}{0.282t + 1} + \frac{1.16(0.2)(2)}{2} v^2 + (692)(9.81) [0.009 + 1.75 \cdot 10^{-6} v^2]$$

$$F_{tr}(t) = \frac{3903}{0.282t + 1} + 61.1 + 0.244 v^2$$

$$F_{tr}(t) = \frac{3903}{0.282t + 1} + 61.1 + 97.6 [\ln(0.282t + 1)]^2$$

$$P_{tr}(t) = F_{tr}(t) v(t)$$

$$= \frac{78060}{0.282t + 1} \ln(0.282t + 1) + 1222 \ln(0.282t + 1) + 1952 [\ln(0.282t + 1)]^3$$



$$1951 \int_0^{10} [\ln(0.282t+1)]^3 dt = \frac{1951}{0.282} \int_1^{3.82} (\ln u)^3 du$$

$$1e+ u = 0.282t + 1$$

$$du = 0.282 dt$$

$$= 6918 \left\{ u(\ln u)^3 - 3 \left[u(\ln u)^2 - 2u \ln u + 2u \right] \right\} \Big|_1^{3.82}$$

$$= 6918 \left\{ 9.20 - 3[6.86 - 10.2 + 7.64] - 0 + 3[0 - 0 + 2] \right\}$$

$$= 6918(2.3) = 15911 \text{ J}$$

$$\int_0^{10} w[F_{AD} + F_{coll}] dt = 9965 + 15911 = 25876 \text{ J}$$

$$\text{so } \Delta l_{tr} = 2.485 \cdot 10^5 + 2.5876 \cdot 10^4$$

$$\boxed{\Delta l_{tr} = 2.744 \cdot 10^5 \text{ J}}$$

$$\frac{\Delta KE}{\Delta l_{tr}} \cdot 100\% = 90.6\%$$

$$\frac{\int_0^{10} w[F_{AD} + F_{coll}] dt}{\Delta l_{tr}} \cdot 100\% = 9.4\%$$

$$(b) \quad \Delta \mathcal{E}_{tr} = \Delta KE + \Delta PE + \int_0^{10 \text{ sec}} v(F_{AD} + F_{roll}) dt$$

$$\Delta KE = \frac{1}{2} m [v^2(10 \text{ sec}) - v^2(0)]$$

$$= \frac{1}{2} (692) (26.8)^2 = 2.485 \cdot 10^5 \text{ J}$$

$$\Delta PE = 0 \quad \text{since road is level}$$

$$\begin{aligned} \int_0^{10 \text{ sec}} v(F_{AD} + F_{roll}) dt &= \int_0^{10 \text{ sec}} (692)(9.81)(0.009) 20 \ln(0.282t+1) dt \\ &\quad + \int_0^{10} \left[692(9.81)(1.75 \cdot 10^{-6}) + \frac{1.16(0.2)(2)}{2} \right] [20 \ln(0.282t+1)] dt \\ &= 1222 \int_0^{10} \ln(0.282t+1) dt + 1951 \int_0^{10} [\ln(0.282t+1)]^3 dt \end{aligned}$$

$$\text{let } u = 0.282t + 1$$

$$du = 0.282 dt$$

$$t = 0 \Rightarrow u = 1$$

$$t = 10 \Rightarrow u = 3.82$$

$$1222 \int_0^{10} \ln(0.282t+1) dt = \frac{1222}{0.282} \int_1^{3.82} \ln u du$$

$$= 4333 [u \ln u - u]_1^{3.82}$$

$$= 4333 [3.82 \ln 3.82 - 3.82 - \cancel{\ln 1} + 1] = 9965 \text{ J}$$

$$F_{tr} = \left[\operatorname{sgn}(V) mg C_0 + mg \sin \beta \right] + \operatorname{sgn}(V) \left[\frac{\rho}{2} C_D A_F + mg C_1 \right] V^2$$

$$F_{tr} = \left[\operatorname{sgn}(V) (692)(9.81)(0.009) + (692)(9.81) \sin \beta \right] + \operatorname{sgn}(V) \left[\frac{1.16}{2} (0.2)(2) + (692)(9.81)(1.75 \cdot 10^{-6}) \right] V^2$$

$$F_{tr} = 61.1 \operatorname{sgn}(V) + 6789 \sin \beta + 0.244 V^2 \operatorname{sgn}(V)$$

if $V > 0$ then

$$F_{tr} = 61.1 + 6789 \sin \beta + 0.244 V^2$$

$$\beta = 0^\circ \Rightarrow F_{tr} = 61.1 + 0.244 V^2$$

$$\beta = 4^\circ \Rightarrow F_{tr} = 535 + 0.244 V^2$$

$$\beta = -4^\circ \Rightarrow F_{tr} = -412 + 0.244 V^2$$

if $V < 0$ then

$$F_{tr} = -61.1 + 6789 \sin \beta - 0.244 V^2$$

$$\beta = 0^\circ \Rightarrow F_{tr} = -61.1 - 0.244 V^2$$

$$\beta = 4^\circ \Rightarrow F_{tr} = 412 - 0.244 V^2$$

$$\beta = -4^\circ \Rightarrow F_{tr} = -535 - 0.244 V^2$$

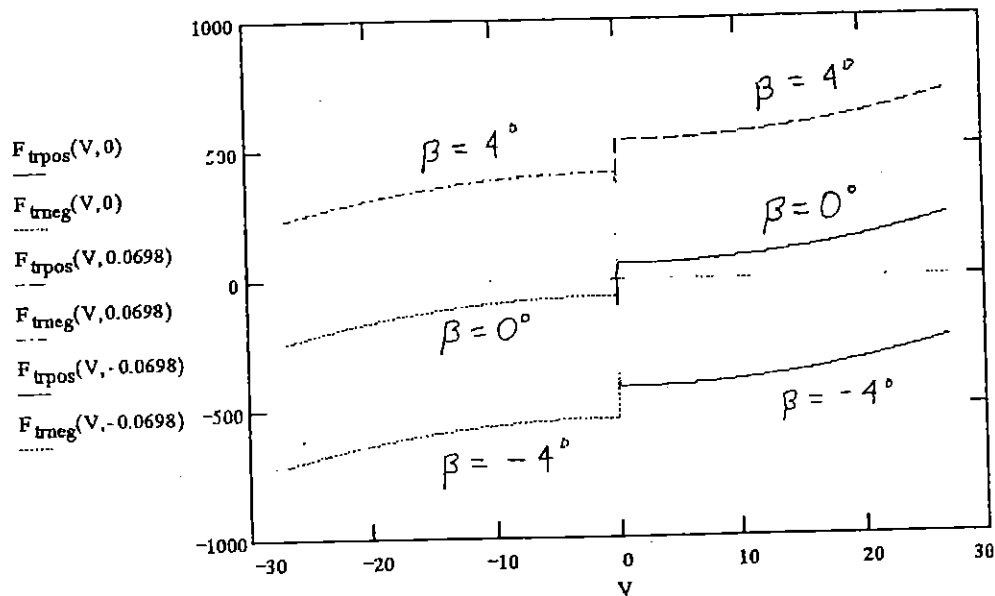
PLOTS FOR 3 STEADY STATE VELOCITIES

$$V := -27, -26.9..27$$

$$F_{trpos}(V, \beta) := (61.1 + 6789 \sin(\beta) + 0.244 \cdot V^2) \cdot \Phi(V)$$

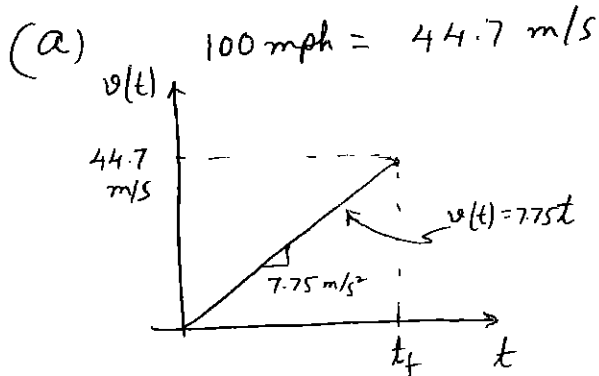
$$F_{trneg}(V, \beta) := (-61.1 + 6789 \sin(\beta) - 0.244 \cdot V^2) \cdot \Phi(-V)$$

Steady-State Tractive Force versus Constant Vehicle Velocity
($\beta = 0, +4, -4$ degrees)



Units: Vert. axis Newtons, Hor. axis m/sec.

PROBLEM 2.7



$$\int_0^{t_f} v \, dt = 100 \text{ m} + \frac{25 \text{ m}}{\cos 30^\circ}$$

$$\Rightarrow \frac{1}{2} t_f \times 44.7 = 100 \text{ m} + 28.9$$

$$\Rightarrow t_f = 5.77 \text{ secs.}$$

(b) $\Delta \text{P.E.} = mg [f(t_f) - f(t_i)] = 692 \times 9.81 \times \left(\frac{25 \tan 30^\circ}{14.43} \right)$

$$= 9.8 \times 10^4 \text{ Joules.}$$

(c) $E_{\text{loss}} = \left[mg c_a \int_0^{t_f} v \, dt + \left(mg c_t + \frac{\rho}{2} C_D A_F \right) \int_0^{t_f} v^3 \, dt \right] = 3.93 \times 10^4 \text{ J}$

ALTERNATIVELY,

$$\Delta E_{\text{TR}} = \Delta \text{K.E.} + \Delta \text{P.E.} + E_{\text{loss}}$$

$$\Rightarrow E_{\text{loss}} = \Delta E_{\text{TR}} - \Delta \text{P.E.} - \Delta \text{K.E.}$$

$$= 8.28 \times 10^5 - .98 \times 10^5 - 6.91 \times 10^5 = 3.9 \times 10^4 \text{ J}$$

$$\Delta \text{K.E.} = \frac{1}{2} 693 \times 44.7^2$$

$$= 6.91 \times 10^5$$

Solution to Problem # 2.8

The vehicle parameters and performance requirements of the vehicle are used for sizing calculations:

Description	Requirements
Vehicle Mass	2000kg
Driver/One passenger	176 Lbs./ 80 kg
Rolling Resistance Coefficient,	$C_0 = 0.01$;
Wheel Radius, r_{wh}	0.3305 m
Aerodynamic Drag Coefficient, C_{AD}	0.45
Frontal Area, A_F	2.5m^2

Mass of driver and 1 passenger,
 $m = 2000 + 2 \times 80 = 2160\text{kg}$

For a grade of 0.5%, $\beta = \tan^{-1} (.5/100) = 0.2865^\circ$

(a) With constant force P_{TR} acceleration

$$\begin{aligned}\frac{P_{TR}}{v} &= m \frac{dv}{dt} + mg[\sin \beta + C_0] + \left[mgC_1 + \frac{\rho}{2} C_D A_F \right] v^2 \\ \Rightarrow \frac{dv}{dt} &= \frac{P_{TR}}{mv} + g[\sin \beta + C_0] + \left[gC_1 + \frac{\rho}{2m} C_D A_F \right] v^2 \quad (\text{P2.81})\end{aligned}$$

(b) Using a Matlab program with equation (1) and given initial conditions at 5 s, and then continuing with constant ($P_{TR} = 145\text{kW}$), the following data s obtained

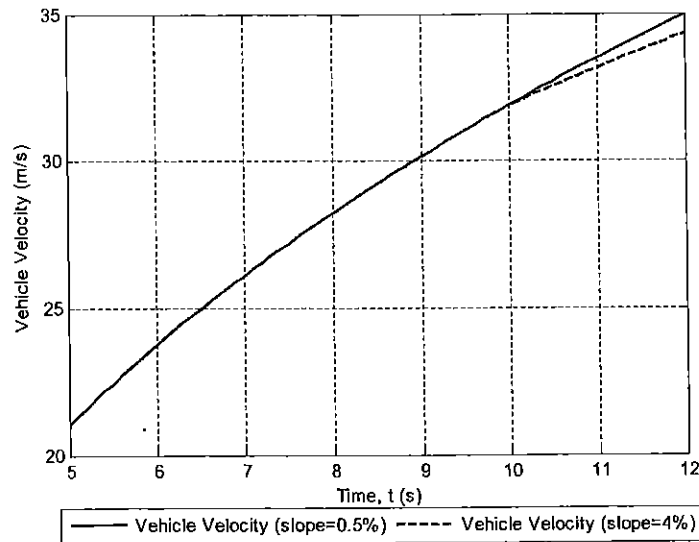
At 10 secs., velocity is 31.99 m/s (= 71.57 mi/h).

The Matlab program used is included at the end of this problem solution

(c) At 12 s, with a slope change to 4% at $t=10$ s, the vehicle velocity at $t = 12$ s is 33.14 m/s (=74.14 mi/h)

If the slope remained at 0.5%, then the vehicle velocity at $t = 12$ s is 34.9686 m/s (=78.23 mi/h)

The plots for both the scenarios is given below:



Matlab program for constant power operation

% Problem 2.8 (b) and (c) of book Solution

clear;

echo off;

Co=.01;

Cd=0.45;

Af=2.5;

g=9.81;

m=2000+160;

row=1.115;

grade=0.5; % 0.5% grade

angle=atan(0.5/100);

% Vehicle accelerates from 0 to 21 m/s in 5 s on a 0.5% grade. Then it accelerates with a

% constant power of 145kW

v_hev(1)=21;% vehicle velocity

b(1)=5; %Time

PTR_const=145000;

PTR(1)=PTR_const;

kkmax=121-50; % Variable for simulation up to 12 s

deltat= 0.1; % step time

for kk=2:kkmax

t=(kk-1)*deltat;

b(kk)=t+5;

```

        v_hev(kk)=v_hev(kk-1)+(PTR_const/(m*v_hev(kk-1))-g*(sin(angle)+Co)-
(0.5*row*Af*Cd*v_hev(kk-1)^2)/m)*deltat;
        FTR(kk)=PTR_const/v_hev(kk); % Traction force
        PTR(kk)=FTR(kk)*v_hev(kk); % Traction Power (constant at 145 kW)

end

v_hev1(1)=21;% vehicle velocity
b1(1)=5; %Time

for kk=2:kkmax
    t=(kk-1)*deltat;
    b1(kk)=t+5;
    if b1(kk)>10
        angle=atan(4/100); %grade changes to 4% at t=10 s
    else
        end
        v_hev1(kk)=v_hev1(kk-1)+(PTR_const/(m*v_hev1(kk-1))-g*(sin(angle)+Co)-
(0.5*row*Af*Cd*v_hev1(kk-1)^2)/m)*deltat;

end

% Plotting speed vs time
plot(b,v_hev,'-',b1,v_hev1,'--','Linewidth',1.5);
grid;

xlabel('Time, t (s)');
ylabel('Vehicle Velocity (m/s)');

legend('Vehicle Velocity (slope=0.5%)','Vehicle Velocity
(slope=4%)','Location','SouthOutside','Orientation','horizontal');
clc;

```

Solution to Problem # 2.9

(a)

The slip s of a vehicle is given

$$s = \left(1 - \frac{\zeta_T v}{\omega_{wh} r_{wh}}\right) \times 100 \% \quad \text{where } \zeta_T \text{ is the total gear ratio from engine/motor to driven}$$

wheels, ω_{wh} is the wheel speed and v is the vehicle velocity.

The vehicle velocity is then given by

$$v = \frac{\omega_{wh} r_{wh}}{\zeta_T} (1 - s) \quad m/s$$

$$\text{The wheel speed is then } \Rightarrow \omega_{wh} = \frac{26.8 * 1}{0.3305(1 - .15)} = 95.4 \text{ rads/sec (assuming } \zeta_T = 1).$$

(b)

The traction force limit is

$$\mu_{tr_limit} = \mu(s) * F_{gvT}$$

$$F_{gvT} \text{ is the normal force and is equal to } mg \cos \beta = 1960 * 9.81 * \cos(.57) = 19,227.6 N$$

(For a grade of 1%, $\beta = \tan^{-1}(1/100) = 0.57^\circ$)

$$\mu(15\%) = \mu_{pk} [a(1 - e^{-bs}) - cs] = 0.85 * [1.1(1 - e^{-20 * .15}) - .0035 * .15] = 0.85 * (1.0452 - .0005) = 0.888$$

for $\mu_{pk} = 0.85$, $a = 1.1$, $b = 20$, and $c = .0035$

Therefore,

$$\mu_{tr_limit} = \mu(15\%) * mg \cos \beta = 0.888 * 19,227.6 = 17,074 N$$